SPIRE

SUBJECT:	Analysis of radiant heating of the SPIRE input filter		
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Change Record

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Simple calculation of SPIRE input filter heating due to absorption of thermal radiation

Assumptions

- Circular filter (should provide representative results for rectangular geometry)
- Periphery of filter is at fixed temperature T_o
- Radiative cooling is negligible (pessimistic)
- Filter thermal conductivity can be represented by a power law
- Thermal conductivity is dominated by the dielectric component (i.e., metalisation does not increase the effective thermal conductivity)

R

t

T,

r

 $k(T) = k_0 T^b$

• Filter is heated by a uniform radiant power density

Definitions

- Filter radius
- Filter thickness
- Filter periphery temperature
- Thermal input to filter (power absorbed per unit area)
- Thermal conductivity law

Consider an annular ring of thickness dr at radius r. Let the temperatures at the inner and outer edges of the ring be T and T + dT respectively.

The thermal conductance across the ring is

$$G(r) = k_0 T^{\beta} \frac{2\pi r t}{dr}$$

In equilibrium, the power flowing across dr is equal to the total power absorbed within radius r, and is

$$W(r) = \pi r^2 \rho$$

The heat balance equation for the ring is

W(r) = G(r)dT

Therefore

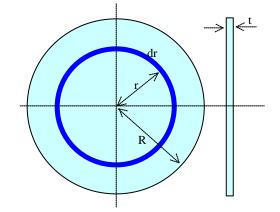
$$\pi r^2 \rho = k_0 T^\beta \frac{2\pi r t}{dr} dT$$

So

$$rdr = \frac{2k_0 t}{\rho} T^\beta dT.$$

Integrating this with the boundary condition T(R) = T_o gives

$$T(r) = \left[\frac{\rho(\beta+1)(R^2 - r^2)}{4k_0 t} + T_0^{\beta+1}\right]^{\frac{1}{\beta+1}}.$$



Calculation of temperature profile as a function of absorbed power density

Fundamental constants:

kb := $1.3806 \cdot 10^{-23}$ h := $6.626 \cdot 10^{-34}$ c := $3 \cdot 10^{8}$ $\sigma := 5.67 \cdot 10^{-8}$

Planck
function
$$B(nu,T) := \frac{2 \cdot h \cdot (nu)^3}{c^2 \cdot \left[e^{\left(\frac{h \cdot nu}{kb \cdot T}\right)} - 1 \right]}$$

 $R := 50 \cdot 10^{-3}$ $t := 1 \cdot 10^{-3}$ Filter radius and thickness (m)

Filter periphery temperature (K) To := 5

Thermal conductivity index $\beta := 0.5$ Thermal conductivity at ko := 0.0131 K (W m-1 K-1)

Representative date from ALW for CTFE from Reed et al. 1973.

Total power falling on filter (mW) $Ptot(\rho) := 1000 \cdot \rho \cdot \pi \cdot R^2$

Temperature vs. radial distance $r := 0, 0.1 \cdot 10^{-3} ... R$

Filter central temperature vs. absorbed power density

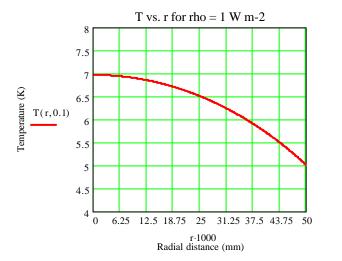
Average filter temperature (K) vs power density (W m-2)

 $Tc(\rho) := \left[To^{(\beta+1)} + \frac{\rho \cdot (\beta+1) \cdot R^2}{4 \cdot k o \cdot t} \right]^{\frac{1}{\beta+1}}$ $\operatorname{Tavg}(\rho) := \frac{\int_0^{\kappa} T(r,\rho) \cdot 2 \cdot \pi \cdot r \, dr}{\pi \cdot P^2}$

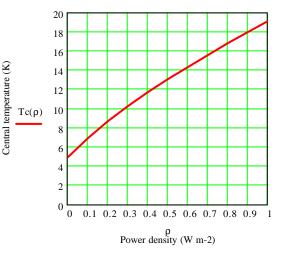
 $T(r,\rho) := \left[To^{(\beta+1)} + \frac{\rho \cdot (\beta+1) \cdot (R^2 - r^2)}{4 \cdot ko \cdot t} \right]^{\frac{1}{\beta+1}}$

Tavg(0.1) = 6.0

T vs. radius for a typical case



Central temp. vs. r $\rho := 0, 0.1..1$



Estimation of heating effect for SPIRE input filter

Filter absorption: assume that the filter reflects 90% of the incident ε filter := 0.1 power across the whole spectrum (estimate from Peter Ade)

1. Power received from the telescope

Focal ratio F := 8.68Telescope: Temperature (K) Ttel := 80 $\Omega := \frac{\pi}{4 \cdot F^2}$ Solid angle subtended by telescope at filter $A\Omega = 8.2 \times 10^{-5}$

 $A\Omega := \pi \cdot R^2 \cdot \Omega$ AW at filter for telescope background (m^2 sr)

Note: We assume here that the flter is well-baffled by the SPIRE FPU structure and sees only the telescope secondary mirror

Define suitable wavelength range (mm) for integration of telescope Planck function

Check

$$\left(\int_{v_1}^{v_2} B(nu, 80) \, dnu\right) \pi = 2.32 \qquad \sigma \cdot 80^4 = 2.32$$

Telescope emissivity vs. wavelength

Assume pessimistically

- * e = 0.04 for all wavelengths above 100 mm
- * e increases with frequency^2 below 100 mm until it hits unity and = unity for frequencies above that point

$$i := 1, 2... 1000$$
 $v_i := 1 \cdot 10^{11} \cdot i$

$$λ1 := 1 \cdot 10^{-6} λ2 := 10000 \cdot 10^{-6}$$

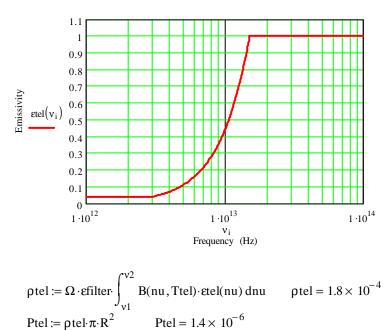
$$v2 := \frac{c}{\lambda 1} v1 := \frac{c}{\lambda 2}$$

$$v2 = 3.0 \times 10^{14} v1 = 3.0 \times 10^{10}$$

$$\epsilon 1(\mathrm{nu}) \coloneqq 0.04 \cdot \left[\frac{\mathrm{nu}}{\left(\frac{\mathrm{c}}{100 \cdot 10^{-6}}\right)}\right]^2$$

$$\varepsilon 2(nu) := if(\varepsilon 1(nu) > 1, 1, \varepsilon 1(nu))$$

$$\varepsilon tel(nu) := if(\varepsilon 2(nu) < 0.04, 0.04, \varepsilon 2(nu))$$



 $Tc(\rho tel) - To = 0.0038$

Power density (W m-2), total power (W) and central temperature (K) of filter when viewing only telescope thermal emission



2. Stray light power received from the cryostat ambient environment in flight

This is estimated (Tony Richards, private communication) as between 0.1 and 1 mW total power. Here we adopt the latter as a pessimistic estimate.

Corresponding power density (W m-2) $\rho stray := \frac{10^{-3}}{\pi R^2}$ $\rho stray = 0.127$ Corresponding filter central
and average temperatures (K) $Tc(\rho stray) = 7.5$ $Tavg(\rho stray) = 6.3$ Values for 1E-4 W total power are 5.3 and 5.1 K
respectively.Values for 1E-4 W total power are 5.3 and 5.1 K

2. Worst case: instrument in ground calibration facility (with no ND or pre-filtering)

Assume the background is a unit emissivity 300-K black body

Power density (W m-2), total power (W) and temperature (K) of filter	$\rho \operatorname{room} := \Omega \cdot \operatorname{efilter} \int_{v_1}^{v_2} B(nu, 300) dnu$		$\rho \operatorname{room} = 1.5 \times 10^{-1}$
	$Proom := \rho tel \cdot \pi \cdot R^2$	$Proom = 1.4 \times 10^{-6}$	Tc(proom) - To = 2.9

Tavg(proom) = 6.5

Comments and conclusions

- * The calculations above are for a 1-mm thick filter 100 mm in diameter. This should give a pessimistic result for the SPIRE input filter (dimensions 120 x 50 mm)
- * The heating effect is dominated by the stray light environment rather than the telescope background power.
- * A stray light power of 1 mW on the filter will result in a temperature rise of around 2.5 K at the centre of the filter and an increase in the average temperature to around 6.3 K. This is envisaged to have a negligible effect on the instrument performance. Analysis using the instrument sensitivity model will be carried out to quantify the effect and to derive a temperature stability requirement for the filter.
- * The above analysis has been carried out using pessimistic assumptions about all of the filter properties. Further refinement of the model is likely to produce lower estimates for the temperature rise.